

Analytical Study of a Modified Thermal Energy Storage System for PS10 in Aswan

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ABSTRACT

According to the need to a renewable energy source for electric power generation to fit the required load curve in the isolated Toshka area near Aswan, it is proposed to install a similar power plant as PS10 of Spain with a modified storage system. PS10 is a commercial CSP of 11MW power that has available analysis in literature. In this study, we compare the performance of the PS10 with a modified storage system when operated in Spain and its performance with the same storage system when operated in Aswan. The introduced storage system has two tanks of suitable molten salts. Such solution is found very promising to accommodate the life in arid zones in Aswan for the whole day-hours. The comparison shows also valuable advantages of selecting Aswan to operate such system in comparison to Spain

Keywords: Solar Power Tower System, CSP, Thermal Energy Storage, Modified Thermal Energy Storage

1. INTRODUCTION

Thermal Energy Storage works to make concentrated solar tower (CST) a more flexible and valuable technology for electricity generation. Thermal Energy Storage makes CST possible to meet electricity peak demand not only by extending the operation time when there is no solar irradiation but also by overcoming weather fluctuations. PS10 is a concentrating solar thermal power plant of 11 MW output power 11 MW. It is based on tower technology for grid-connected electricity generation [1, 2, 3, 4, and 5]. PS10 plant is located in the town of Sanlúcar la Mayor 37.2° Latitude, 25 km west from the city of Seville. PS10 plant is working with Direct Saturated steam Generation (DSG) concept, at considerably low temperatures (250°C – 270 °C), at a pressure of 40bar. The annual energy generation amounts to 23GWh as presented in Figure (1) [6, 7].

MPS10 plant is a PS10 plant with two tank molten salt thermal energy storage. In a molten-salt solar power tower, liquid salt at 290°C (minimum operation temperature) is pumped from a ‘cold’ storage tank through the receiver where it is heated to 565°C (maximum operation temperature) [8, 9, 10, and 11] and then on to a ‘hot’ tank for storage. When power is needed from the plant, hot salt is pumped to a steam generating system that produces superheated steam for a conventional Rankine cycle turbine/generator system. From the steam generator, the salt is returned to the cold tank where it is stored and eventually reheated in the receiver [12, 13]. The molten salt is a nitrate salt consisted by (60%NaNO₃, 40%KNO₃) with a specific heat (C_p) equal to (2660J/Kg.K.) and a density of (1870kg/m³) at (565°C). These values will be considered constant during the calculations for different operating temperatures [4, 14]. Figure (2) is a schematic diagram of the primary flow paths in a two tank molten-salt solar power plant.

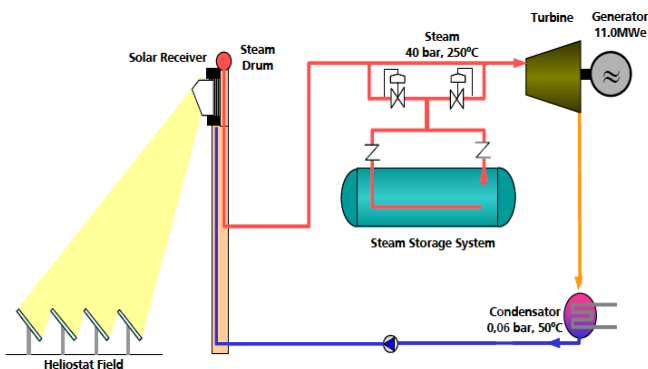


Figure (1) PS10 in Sanlúcar la Mayor, Spain

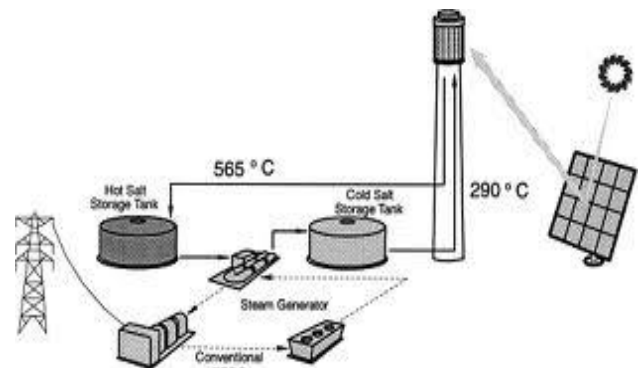


Figure (2) two tank molten salt solar power plant

2. STORAGE SYSTEM

It's possible to design the storage system, consisted by hot tank and cold tank and using dynamic differential equations for the heat transfer between the fluid and each tank, with energy balances on the cold and the hot tanks. The followed criteria to design the storage system are based on the autonomy of the CSP plant, where the goal is to achieve many hours of electrical power production per day without solar radiation. First of all it's assumed that the system doesn't have any losses ($U=0.0$; adiabatic operation) i.e. Outlet tower temperature ($T_{tr,o}$) is equal to the hot tank outlet temperature ($T_{h,o}=565^{\circ}\text{C}$), and the outlet temperature of heat exchanger at steam generator block ($T_{tur,o}$) is equal to the outlet temperature of the cold tank ($T_{c,o}=290^{\circ}\text{C}$)[4]. Second point in this work was the mass flow rate of molten salt to heat exchanger of power block (\dot{m}_{tur}) is constant along the turbine operation. But the flow rate of molten salt to the tower (\dot{m}_{tr}) is varying to remains the outlet tower temperature ($T_{tr,o}$) constant at (565°C) by using a control on the pumping system of the tower. The energy balance equations applied to the MPS10 storage plant as the following [2, 4, 15 and 16].

2.1 Tower

The formula which describes the tower receiver operation can be expressed as follows [4]:

$$\eta_{tr} = \frac{\dot{m}_{tr} Cp (T_{tr,o} - T_{tr,in})}{Q_{tr}} \quad (1)$$

2.2 The Hot storage tank

The following equations-set describes the energy and mass balance of the hot storage tank [4, 15]:

$$\frac{dM}{dt} = \dot{m}_{tr} - \dot{m}_{tur} \quad (2)$$

$$\frac{dE}{dt} = \dot{m}_{tr} Cp T_{h,in} - \dot{m}_{tur} Cp T_{h,o} \quad (3)$$

$$\frac{dE}{dt} = (\dot{m}_{tr} - \dot{m}_{tur}) Cp T_{h,o} + M Cp \frac{dT_{h,o}}{dt} \quad (4)$$

2.3 The Steam generator

The following equation expresses the steam generation in the boiler [4, 17]:

$$Q_{sg} = \dot{m}_{tur} Cp (T_{tur,in} - T_{tur,o}) \quad (5)$$

2.4 The Cold storage tank

The following equations-set describes the energy and mass balance of the cold storage tank which is similar to the hot tank in those equations [4, 15]:

$$\frac{dM}{dt} = \dot{m}_{tur} - \dot{m}_{tr} \quad (6)$$

$$\frac{dE}{dt} = \dot{m}_{tur} Cp T_{c,in} - \dot{m}_{tr} Cp T_{c,o} \quad (7)$$

$$\frac{dE}{dt} = (\dot{m}_{tur} - \dot{m}_{tr}) Cp T_{c,o} + M Cp \frac{dT_{c,o}}{dt} \quad (8)$$

4. RESULTS

The electrical power for PS10 is 11MWe that required 30.17MWe at the tower receiver. If this power is constant during the daylight hours (11 hours average for Spain) this means that the capacity factor is 45.8% and this can show clearly by the simulation results on the MATLAB program. Figure (3) shows the outlet temperature of the hot tank and the outlet temperature of the power block. Figure (4) shows the behaviour of mass of the hot and the cold tank the first days of plant operation.

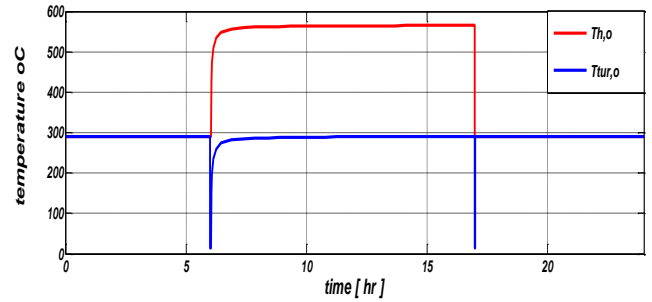


Figure (3) Simulation results of outlet temperature of the hot tank and the outlet temperature of the power block

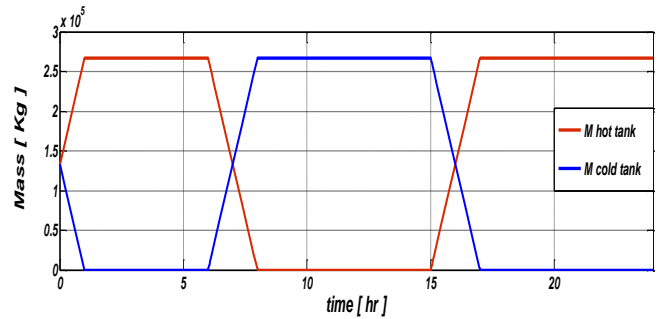


Figure (4) Simulation results of the mass of the cold and hot tank at the first day of plant operation

Now we should consider the actual condition of solar flux along the days of the year for MPS10 plant in Spain and Egypt. In this analysis via an Excel sheet we take 21th March as a reference day for plant operation.

4.1 Results of MPS10 in Spain

MPS10 plant in Spain with total optical efficiency of 66.74% [13] required (400Ton) of molten salt besides two storage tanks of dimensions (213.9m³, 4m height, and 8.25m diameter). This increases the capacity factor of the plant to (0.375) i.e. 90MWh during the day. Figure (5) show solar and plant power during 21th, 22th march. Figure (6) show the mass of hot and cold tanks during the beginning of operation and figure (7) show the next two days.

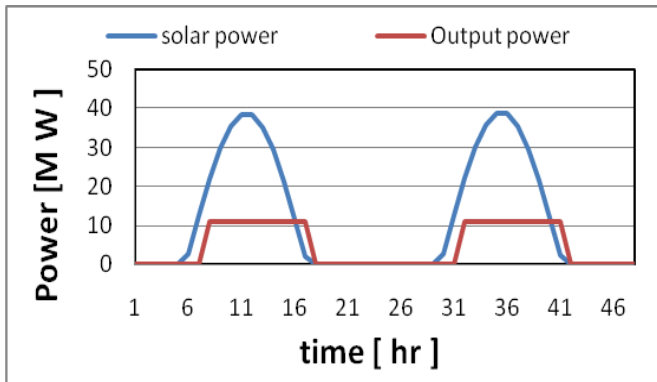


Figure (5) The solar and plant power during 21th, 22th march for MPS10 in Spain (37.5 % capacity factor)

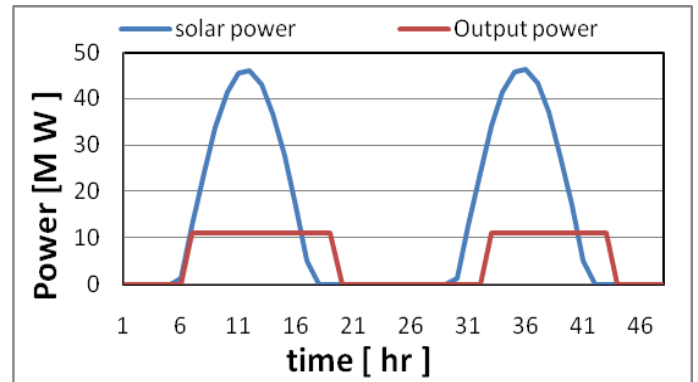


Figure (8) The solar and plant power during 21th, 22th march for MPS10 in Aswan (45.8% capacity factor)

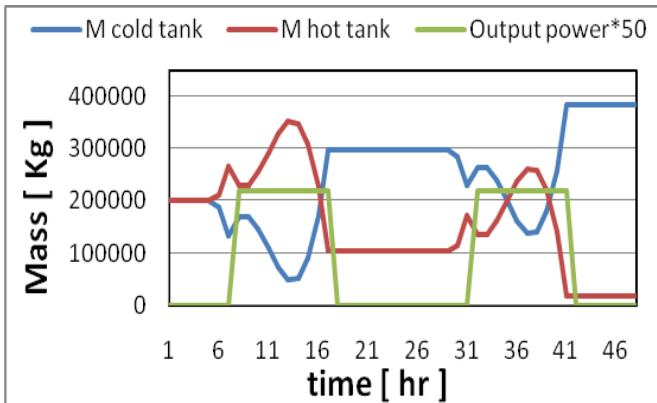


Figure (6) The behaviour of mass of hot and cold tanks and the electrical power generation during 21th, 22th march for MPS10 in Spain (37.5% capacity factor)

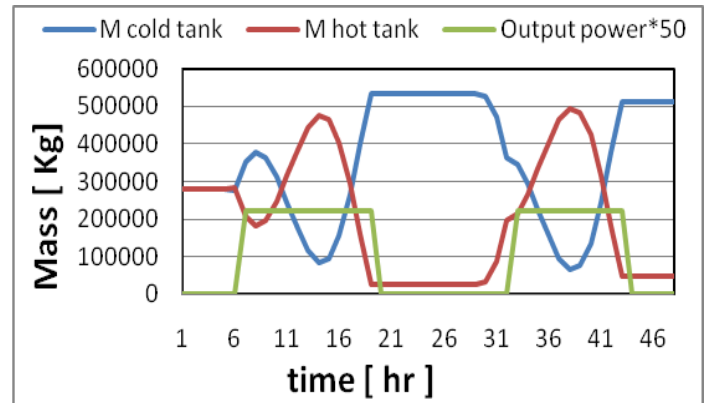


Figure (9) the behaviour of mass of hot and cold tank and the electrical power generation during 21th, 22th march for MPS10 in Aswan (45.8% capacity factor)

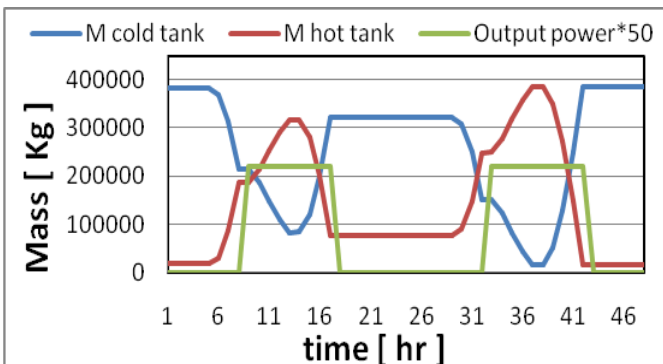


Figure (7) The behaviour of mass of hot and cold tank and the electrical power generation during 23th, 24th march for MPS10 in Spain (37.5% capacity factor)

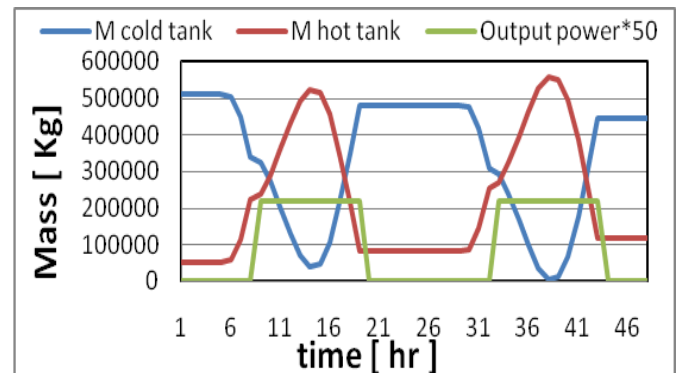


Figure (10) the behaviour of mass of hot and cold tank and the electrical power generation during 23th, 24th march for MPS10 in Aswan (45.8% capacity factor)

4.2 Results of MPS10 in Egypt

MPS10 plant in Aswan with total optical efficiency of 67.47 % [13] required (560Ton) of molten salt besides two storage tanks of dimensions (299.5m³, 4m height, and 9.76m diameter). This increases the capacity factor of the plant to (0.458) i.e. 110MWh during the day. Figure (8) show solar and plant power during 21th, 22th march. Figure (9) show the mass of hot and cold tank during the beginning of operation and figure (10) show the next two days.

5. FEASIBILITY STUDY OF MPS10 IN ASWAN

PS10 in Spain was built in December 2005 and opened in March 2007. The total Cost is (\$47M), the specific cost is about 4200\$/kWgross. In order to calculate the capital cost of the MPS10 for Spain and Aswan in 2013, the inflation rates are taken from 2007 to 2013 [18, 19, 20, 21, and 22]. The total calculated cost of the PS10 for Spain and Aswan in 2013 is (\$50.6M), the specific cost is about 4600\$/kWgross. Figure (11) show the total cost of the plant with increasing capacity factor of the MPS10 plant.

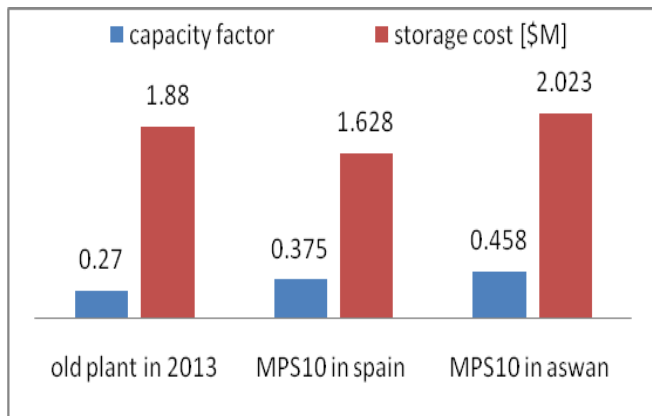


Figure (11) increasing capacity factor against storage cost for MPS10

The landing cost of the MPS10 (\$1M) is ignored in Toshka, Aswan due to the governmental support for encourages national energy project on desert lands. The indirect cost (\$8.6M) also is reduced in Toshka, Aswan due to the relatively low cost of labour and transportation. In Europe, about (\$2.2 M) of the capital cost yearly goes for operating and labours. In Aswan, the yearly operating and labour cost is reduced by 75% (equal about 0.5M \$/year).

6. CONCLUSIONS

The main conclusions come out of this work are summarized as follow:

- The PS10 plant in Spain is used as a demonstrative model in order to estimate the relative advantage of installing a modified storage one in Toshka, Aswan.
- PS10 with two tanks molten salts thermal energy storage system (MPS10) in Spain increasing the capacity of the existing PS10 by (25MWh) daily.
- The daily capacity of MPS10 in Aswan (110MWh) is higher than the capacity of the MPS10 in Spain (90MWh).
- The capital cost of MPS10 in Spain (\$50.348M) is less than the capital cost of the existing PS10 in Spain (\$50.6M). And the Storage cost of MPS10 in Spain (\$1.628M) lowers than the storage cost of MPS10 in Aswan (\$2.023M).
- The feasibility study shows that installing MPS10 in Toshka saving about (\$7.9M) labour and indirect cost. And helping for accommodate the life in it.

NOMENCLATURE

CST: Concentrated Solar Tower.
 C_p : Specific Heat at constant Pressure
 (2660 J/Kg.k for solar salt).
 DSG: Direct Saturated steam Generation.
 M : Mass of tank (Kg).
 \dot{m}_{tr} : Tower flow rate (Kg/s).
 \dot{m}_{tur} : Steam generator flow rate (Kg/s).
 Q_{sg} : Steam generator power (Watt/m²).

Q_{tr} : Tower Receiver Irradiation (Watt/m²).
 $T_{tr,o}$: Tower outlet temperature.
 $T_{tr,in}$: Tower inlet temperature.
 $T_{h,in}$: Hot tank input temperature.
 $T_{h,o}$: Hot tank outlet temperature.
 $T_{tur,o}$: Steam generator outlet temperature.
 $T_{tur,in}$: Steam generator inlet temperature.
 $T_{c,in}$: Cold tank inlet temperature.
 $T_{c,o}$: Cold tank outlet temperature.
 U : Overall heat transfer coefficient.
 V : Tank Volume (m³).
 dM/dt : Change of the tank mass.
 dE/dt : Change of tank total energy.
 η_{th} : Thermal efficiency of Rankine cycle (45%).
 η_{tr} : Tower receiver efficiency (90%).
 ρ : Density of heat transfer fluid
 (1870Kg/m³) for solar salt.

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